

## FSW NUMERICAL SIMULATION OF ALUMINIUM PLATES BY SYSWELD - PART I

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**Abstract:** Friction Stir Welding (FSW) is one of the most effective solid state joining processes and it has numerous potential applications in many industries. The simulation process can provide the evolution of physical quantities such as temperature, metallurgical phase proportions, stress and strain which can be easily measured during welding. The numerical modelling requires the modelling of a complex interaction between thermal, metallurgical and mechanical phenomena. The aim of this paper is to describe the thermal-fluid simulation of FSW using the finite element method. In the theoretical part of the paper heating is provided by the material flow and contact condition between the tool and the welded material. The thermal-fluid results from the numerical simulation for aluminium alloy using SYSWELD are also presented in this paper.

**KEYWORDS:** Friction Stir Welding (FSW); aluminium alloy; Finite element method; thermo-fluid model

### 1 Introduction

Friction stir welding (FSW) is a relatively new joining technology which was developed and patented in 1991 by The Welding Institute (TWI), in the United Kingdom [1]. This is a solid state welding process providing good quality of butt and lap joints. The FSW process has been proved to be ideal for creating high quality welds in a number of materials including those which are extremely difficult to weld by conventional fusion welding. A schematic of friction stir welding process is illustrated in Fig. 1.

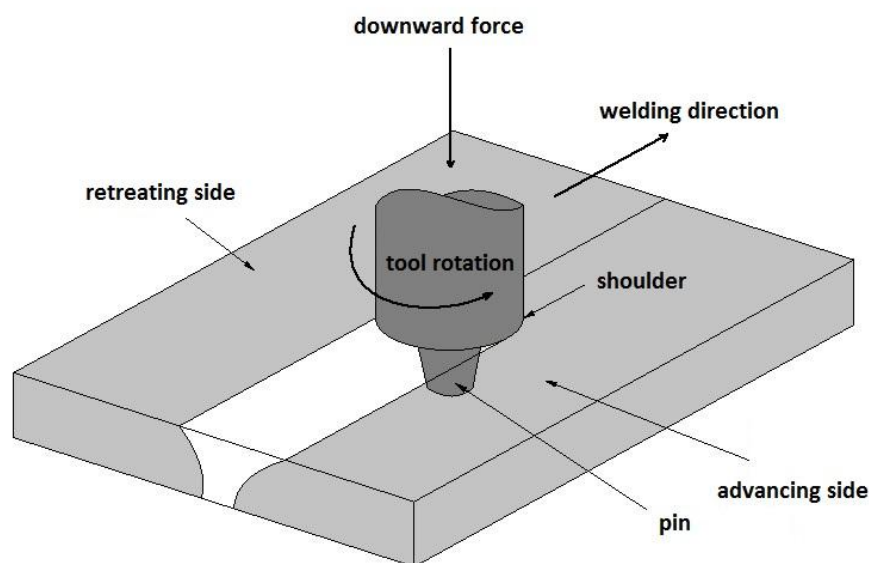


Fig. 1. Schematic diagram of FSW

The welding zone is completely isolated from the atmosphere during the welding process. The advantages of this type of welding are minimized formations of voids in the welding zone, so that welding defects and large distortions commonly associated with fusion welding are minimized or avoided. This welding technique is extensively applied to the aerospace, automobile and shipbuilding industries [5, 6, 7, 8, 9, 10].

## 2. Theoretical background

In FSW, the heat generation takes place mainly under the shoulder, leading to a variable thermal gradient in the through-thickness direction of the plate. The general three dimensional partial differential equation of heat conduction in solid can be represented by

$$\frac{\partial(\rho c T)}{\partial t} = \frac{\partial}{\partial t} \left( k_x \frac{\partial T}{\partial x} \right) + \frac{\partial}{\partial t} \left( k_y \frac{\partial T}{\partial y} \right) + \frac{\partial}{\partial t} \left( k_z \frac{\partial T}{\partial z} \right) + \frac{q_o}{V}, \quad (1)$$

where  $\rho c$  is the volume heat capacity;  $x$ ,  $y$  and  $z$  are the space coordinates;  $k_x$ ,  $k_y$  and  $k_z$  are thermal conductivity,  $T$  temperature,  $t$  time and  $q_o/V$  is the source term.

### 2.1 Heat generation during FSW

For the ideal case considered in Fig.1, the torque required to rotate a circular shaft relative to the plate surface under the action of an axial load is given by [2]

$$M = \int_0^{M_R} dM = \int_0^R \mu P(r) 2\pi r^2 dr = \frac{2}{3} \mu \pi P R^3, \quad (2)$$

where  $M$  is the interfacial torque,  $\mu$  is the friction coefficient,  $R$  is the surface radius, and  $P(r)$  is the pressure distribution across the interface. If all the shearing work at the interface is converted into frictional heat, the average heat input per area and time becomes [2]

$$q_o = \int_0^{M_R} \omega dM = \int_0^R \omega 2\pi \mu P r^2 dr, \quad (3)$$

where  $q_o$  is the net power (in Watts) and  $\omega$  is the angular velocity (in rad/s). Angular velocity can be expressed in term of the rotation speed  $N$ . By substituting  $\omega = 2\pi N$  into Eq. (3), we get

$$q_o = \int_0^R 4\pi^2 \mu P N r^2 dr = \frac{4}{3} \pi^2 \mu P N R^3. \quad (4)$$

From Eq. (4), it is obvious that the heat input depends both on the applied rotational speed and the shoulder radius, leading to a non-uniform heat generation during welding.

### 2.2. Heat generation in SYSWELD by FSW modulus

In SYSWELD by FSW modulus is effective viscosity  $\mu$  defined from Newton-Hoff law used to model the hot forming process [3]:

$$\mu = K(T) \cdot (\sqrt{3} \cdot D)^{m(T)-1}, \quad (5)$$

where  $K$  and  $m$  are the consistency and the sensibility of material, and  $D$  is the equivalent strain rate

$$D = \sqrt{\frac{2}{3} \bar{D} : \bar{D}} \quad (6)$$

The friction between the toll and the workpiece is of Neumann boundary condition. Heat flux density modelled between tool and material contact is [3]

$$q = \beta \cdot \vec{\tau} \cdot \Delta \vec{v}, \quad (7)$$

where  $\beta$  is the proportion of the heat dissipated thought viscous dissipation  $\vec{\tau} \cdot \Delta \vec{v}$  and received by the sheets. All thermal boundary conditions are in the Fig. 2.

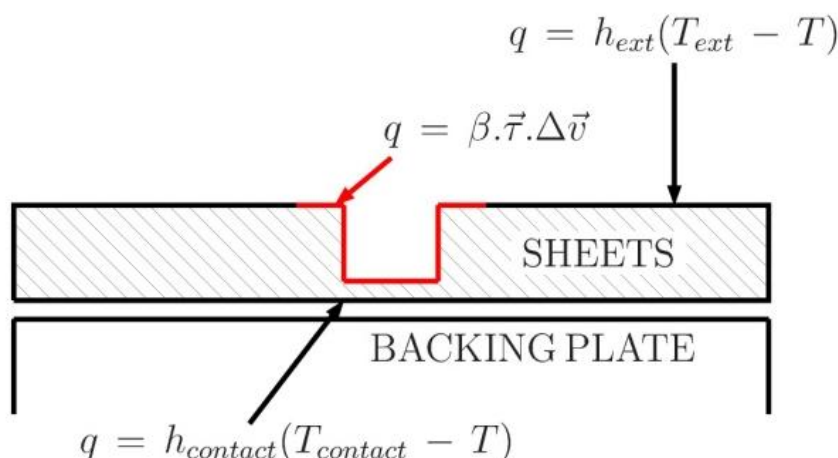


Fig. 2. Thermal boundary conditions

### 3. Numerical solution by SYSWELD

An example is presented in this section. The thermal properties for sheet and backing plate are described in Tab. 1, the friction coefficient is 0.238, linear welding velocity is 1.67 mm/s, tool rotation velocity 41.89 rad.s<sup>-1</sup>, room temperature 15 °C and heat exchange coefficient for convection 19 W/(m<sup>2</sup>.K). A finite element model of sheet and backing plate is presented in Fig. 3. A finite element model of tool is shown in Fig. 4. The results of the solution from software SYSWELD® are presented in fig. 5 and 6.

Table 1. Material properties of sheet and backing plate.

T [°C]	Sheet (Aluminium alloy)			Backing Plate (Steel)		
	$k \left[ \frac{\text{W}}{\text{mm} \cdot ^\circ\text{C}} \right]$	$\rho \cdot 10^{-6} \left[ \frac{\text{kg}}{\text{mm}^3} \right]$	$c \left[ \frac{\text{J}}{\text{kg} \cdot ^\circ\text{C}} \right]$	$k \left[ \frac{\text{W}}{\text{mm} \cdot ^\circ\text{C}} \right]$	$\rho \cdot 10^{-6} \left[ \frac{\text{kg}}{\text{mm}^3} \right]$	$c \left[ \frac{\text{J}}{\text{kg} \cdot ^\circ\text{C}} \right]$
20	0.130	2.750	898.0	0.068	7820	430
120		2.730	951.0			500
220		2.710	1003.0	0.059	7800	550
320		2.690	1055.0			580
420		2.660	1108.0	0.047	7730	610
500						650
587	0.170	2.630	1195.0			
600				0.036	7653	710
644		2.450	1200.0			
700					7613	790
800				0.029		865
900				0.027		565
1450				0.033		
1600					7190	630
2447		2.230	1300.0			707

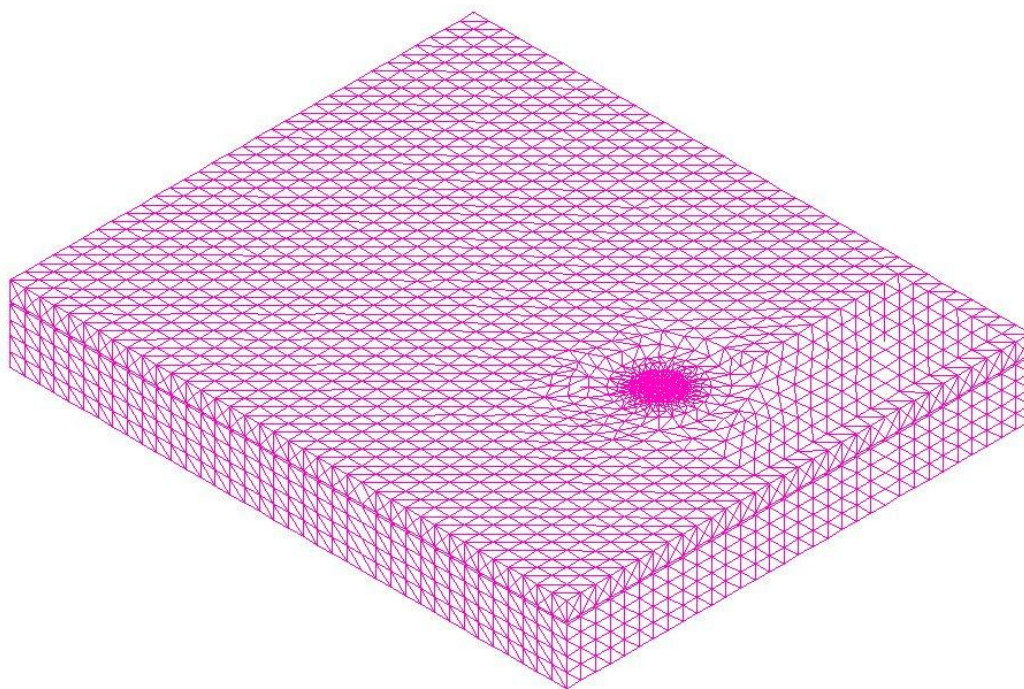


Fig. 3. FEM model of sheet and backing plate

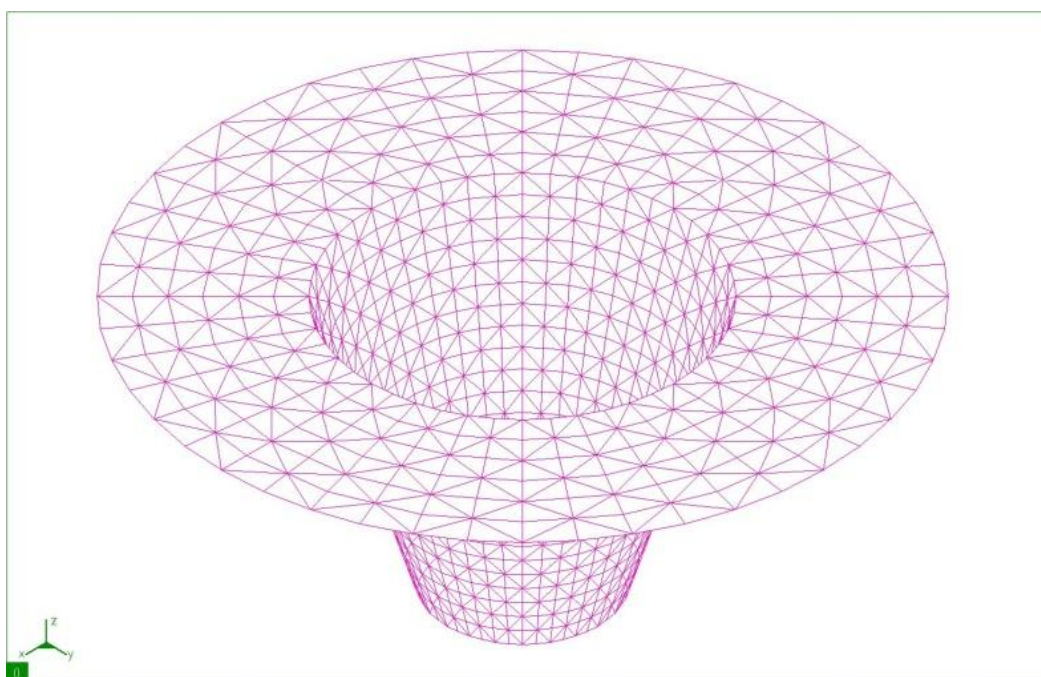


Fig. 4. FEM model of tool



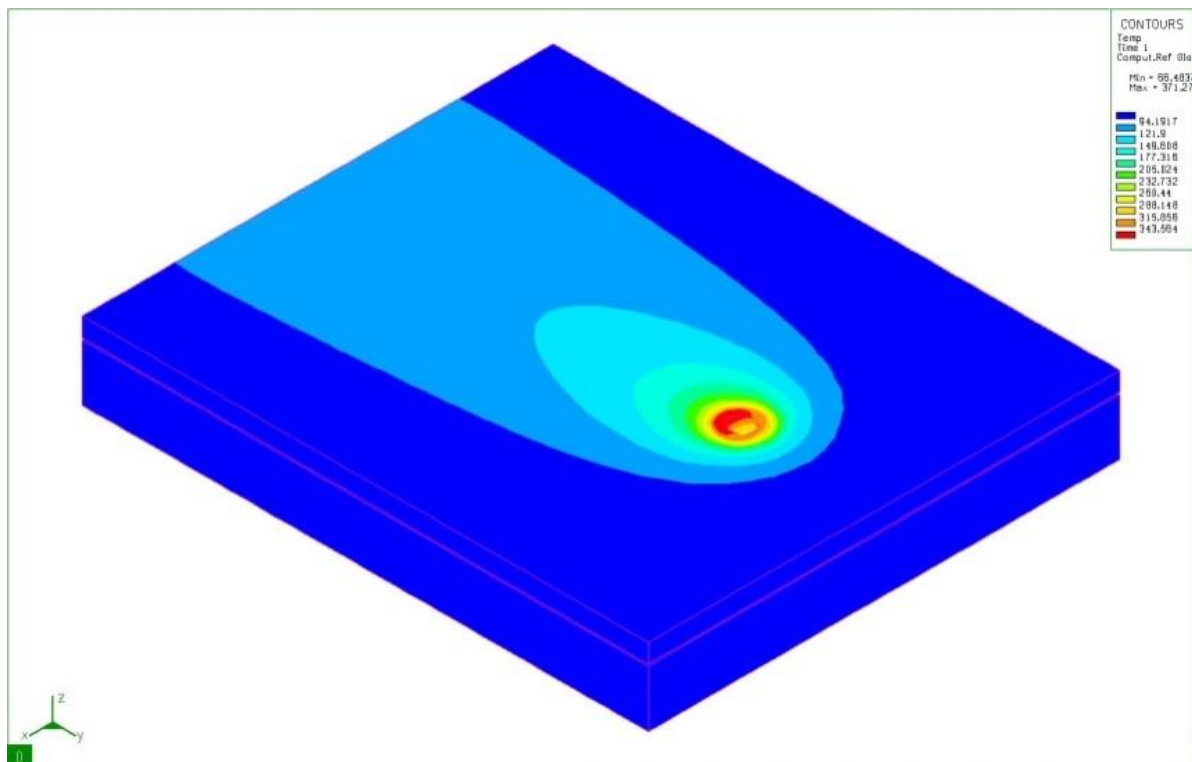


Fig. 5. Temperature profile

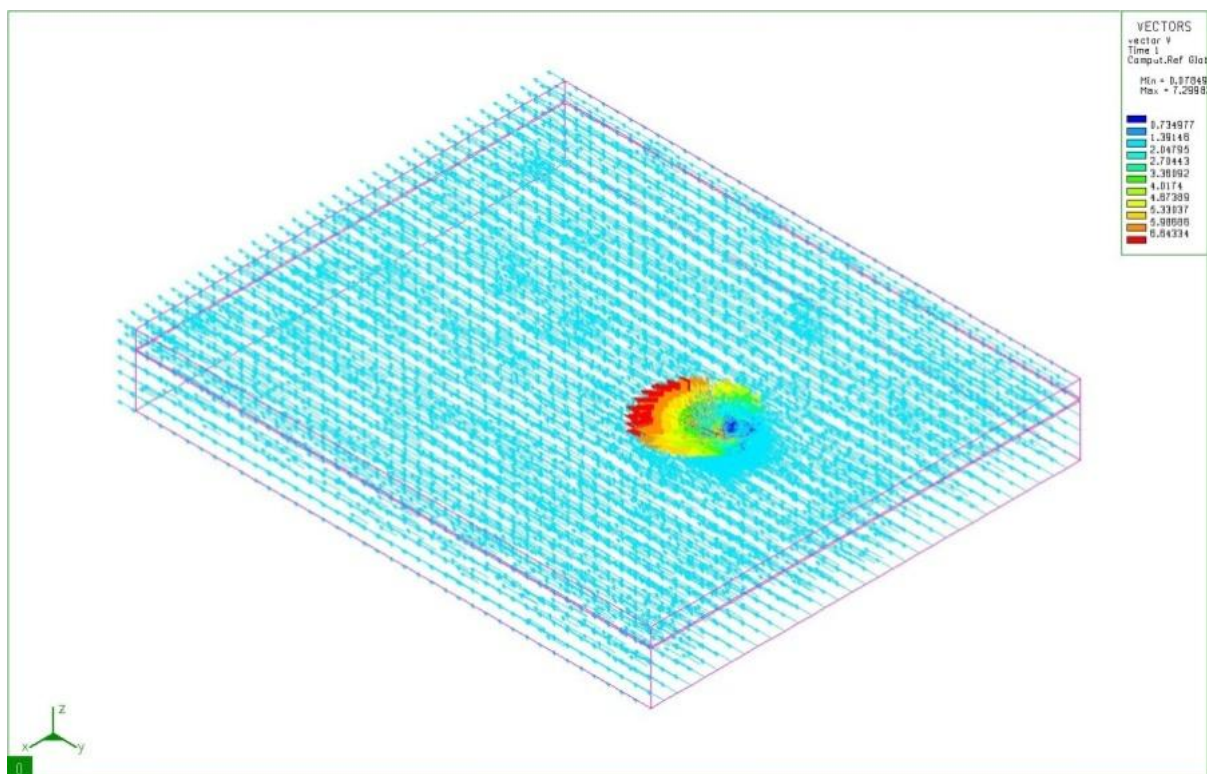


Fig. 6. Velocity field

#### 4. Conclusion

In this paper, a 3D finite element procedure is presented to model the thermo-fluid flow in FSW for the stationary step in SYSWELD®. For the computation of the example proposed in section 3, the mesh is composed of 13457 nodes and 69111 elements. In Fig. 5 the temperature field from SYSWELD® is presented. The numerical results were compared with experimental measurement by

thermo-camera and thermocouples. In section 2 is presented theoretical background for solution of FSW. The results have proved, as it was expected, that high temperature gradients are created. As a result, a further study ought to be carried out to investigate how considering the 2<sup>nd</sup> sound phenomenon [4] in the mathematical model affects the temperature field.

## Acknowledgements

This publication is the result of the project implementation: Research of friction stir welding (FSW) application as an alternative to melting welding methods no. 26240220031 supported by the Research & Development Operational Programme funded by the ERDF.

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