Small Scale Solar Cooling Unit in Climate Conditions of Latvia: Environmental and Economical Aspects

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Abstract – The paper contributes to the analyses from the environmental and economical point of view of small scale solar cooling system in climate conditions of Latvia. Cost analyses show that buildings with a higher cooling load and full load hours have lower costs. For high internal gains, cooling costs are around 1,7 €/kWh and 2,5 €/kWh for buildings with øwer internal gains. Despite the fact that solar cooling systems have significant potential to reduce CO₂ emissions due to a reduction of electricity consumption, the economic feasibility and attractiveness of solar cooling system is still low.

Keywords – absorption chiller, cooling load, reduction of CO_2 , solar collector, solar cooling.

I. INTRODUCTION

It is estimated that the building sector (residential and tertiary) accounts for about 40% of energy consumption worldwide. About one third of the consumption is related to cooling or air conditioning in buildings, especially in southern climates. 80% of the greenhouse gas emissions in Europe still come from the energy sector [1]. The demand for building cooling has been increasing in the past few years and will continue to increase. There are several reasons for such a forecast: more and more popular tendency of modern architecture to use glass facade surface areas, higher demand for comfort, quantitative increase of office and service buildings, increasing number of electric appliances and, in one word, expected economic growth [1,3]. The presence of solar irradiation and the need for cooling practically occur simultaneously, which makes solar cooling an attractive alternative to conventional (basically electric driven) cooling in modern buildings [4]. All that implies energy, economic and environmental consequences because solar cooling systems use less primary energy than conventional cooling systems thus emit less CO₂ and have less impact on the environment [5].

Compared with the traditional compressor-based air conditioner, the solar cooling system can save up to 80% electric energy when providing the same cooling capacity for office buildings. Hence, the system offers a good energy conservation method for office buildings [6, 7, 8]. Solar fractions therefore need to be higher than about 50% to start saving primary energy [9].

Absorption is the most commonly used type of thermally driven cooling unit. Approximately 60% of these systems use absorption chillers, 11% adsorption chillers, and 29% use open systems (desiccant and liquid sorption systems). The solar cooling capacity is assisted by solar flat plate collectors or solar vacuum tube collectors [10,11] or by waste heat. For the often used single effect cooling machines, the ratio of cold production to input heat (COP) is in the range of 0.5-0.8, while electrically driven compression chillers today work at COPs around 3.0 or higher [12,13].

For much of Europe, increases in cooling energy demand due to global warming will be outweighed by reductions in the need for heating energy [14]. But solar cooling systems are still expensive. Investment costs vary between $3200-5100 \in$ per kilowatt of cooling output for small scale systems [15, 16]. The reasons for this are: lack of standardization of the system configurations and mass production, as well as lack of practical experience.

The cooling load of a building depends on climate conditions, thermophysical properties and parameters (insulation thickness, thermal conductivity etc.) of the building envelope, and internal gains (inhabitants, electrical appliances etc.) [17-25]. That is why it is necessary to consider, on one hand, energy efficiency measures and on the other hand - the appropriate and environmentally friendly cooling technologies to cover existing cooling loads of buildings. A long runtime of a solar cooling system have several advantages when viewed from the perspective of environmental aspects as well as from economical aspects.

II. SCOPE AND METHODOLOGY OF THE PAPER

The paper analyses the environmental and economical aspects of small scale solar cooling system in climate conditions of Latvia. The goals are to specify the associated costs and calculate the reduction of CO_2 in comparison with a traditional cooling unit to cover cooling load of the building.

To evaluate environmental and economical aspects of small scale solar cooling systems, cooling load files were produced on an hourly basis with a developed cooling load calculation model *CooL* for a small existing building with about 520 m² total cooled area, 120 m² total windows area (67 m² old wooden frame and 53 m² plastic frame windows), and 346 m² façade area. The air exchange rate was 0.545 h⁻¹. Specific heat losses via enclosed constructions of building were 1072.2 W/K, but total specific heat losses of building included ventilation, doors, roof, walls and windows were 1509.8 W/K. [26]. The building internal gain and characteristics of the building envelope were varied to analyze the economical and

environmental aspects of a solar cooling system. The building internal gain include all heat from electrical equipment and inhabitants. To cover the cooling load of the building for a solar cooling system, a commercially available absorbtion chiller with 15 or 25 kW nominal cooling power was chosen.

The cooling load of the absorption chiller was chosen to maintain internal room temperature levels at a given set point of 24°C for all operation hours.

III. EXPECTED BUILDING COOLING LOAD DETERMINATION

To evaluate the environmental aspects and economic performance of solar cooling systems under varying conditions, different building cooling load files were produced with the calculation tool *CooL*.



Fig. 1. Existing building with high internal loads (Build1_High).

The air exchange rates were held constant at 0.5 h^{-1} for the building throughout the year. Surplus heat from the building analyzed was removed by a water-based distribution system, which was fed cold water from the cooling unit, to maintain a set internal temperature.

The power of the absorption cooling unit was chosen to keep the internal temperature levels below a set point of 24°C for all operation hours (see Table II).

To evaluate the economic and environmental performance of the solar cooling system, four different energy consumption and cooling energy demand profiles were produced for the existing building using the calculation tool CooL:

- existing building with low internal gains of 4 W/m² – acronym Build1_Low,
- existing building with high internal gains of 20 W/m² – acronym Build1_High (see Fig. 1),
- building after implementation of energy efficiency measures with low internal gains of 4 W/m² – acronym Build2_Low (see Fig. 2),
- building after implementation of energy efficiency measures with high internal gains of 20 W/m² – acronym Build2_High,
- reference building in Stuttgart (Germany) with low internal gains of 4 W/m² acronym Build3_Ref.



Fig. 2. Building with low internal loads and improved efficiency (Build2_Low).



Fig. 3. Specific and total annual cooling energy demand for the building with different internal gains.

| THE SELECTION AND FARAMETERS OF THE BUILDING FOR COULING EOAD CALCULATIONS | | | | | | | | | |
|--|--|---|--|--|--|--|--|--|--|
| Building case | Internal gains (W/m ²) | Overall heat transfer coefficient of the walls, U_{wall} (W/m ² K) | Overall heat transfer coefficient of the windows, U_{win} (W/m ² K) | Overall heat transfer coefficient of the roof, U_{roof} (W/m ² K) | Overall heat transfer coefficient of the floor U_{roof} (W/m ² K) | | | | |
| Build1_Low | 4 | 1.07 | 1.9-2.38 | 0.68 | 0.67-1.33 | | | | |
| Build1_High | 20 | 1.07 | 1.9-2.38 | 0.68 | 0.67-1.33 | | | | |
| Build2_Low | 4 | 0.4 | 1.8 | 0.35 | 0.45 | | | | |
| Build2_High | 20 | 0.4 | 1.8 | 0.35 | 0.45 | | | | |

TABLE I THE SPECIFICATIONS AND PARAMETERS OF THE BUILDINGS FOR COOLING LOAD CALCULATIONS

The cooling energy demand of building is between 400 kWh in the building after implementation of energy efficiency measures with low internal gains up to 3400 kWh in the

existing building with high internal gains (see Fig. 3). A wide range of specific cooling energy demands are covered, ranging from about 0.8 kWh/m^2 annual for a building after energy

efficiency measures with low internal gains up to 6.4 kWh/m^2 per annum for the existing building with high internal gains.

The specifications and parameters for the existing building and building after energy efficiency measures are summarized in Table I.

IV. SOLAR COOLING SYSTEM DESCRIPTION AND CONFIGURATION

For an annual cooling energy demand from 400 up to 3400 kWh and average COP of 0.55 the cooling unit whole system requires from 720 to 6100 kWh of heating energy. The design generator entry temperature is 65° C at an absorber cooling entry temperature of 26° C and an evaporator entry of 12° C and exit of 6° C. To achieve a solar fraction of at least 60% for the given cooling load profiles, a collector surface area of 2 to 26 m^2 and a storage tank volume of 0.2 up to 2.1 m³ is required, if the generator is always operated at a mean temperature of 65° C.



Fig. 4. Schematic view of the solar single effect absorbtion cooling system.

For the constant generator inlet temperature level of 65° C, the specific collector yield is in the range from 237 up to 370 kWh/m² per year for an annual solar irradiance of 1080 kWh/m². The solar thermal system efficiency was assumed to be 22% (for flat plate solar collectors) and 34% (for vacuum solar collectors), respectively.

The size of the solar thermal system is based on the capacity of the absorbtion cooling unit and available solar irradiation. The solar cooling system design values for the different buildings are summarized in Table II. A schematic view of the solar single effect absorption cooling system is shown in Figure 4.

A. Solar thermal system

The solar cooling system model includes a solar collector (flat plate and vacuum tube), a stratified storage tank, an absorbtion chiller, a control unit, and a back-up heater. A solar collector is modeled using the steady-state solar collector equation with an optical efficiency η_0 and the two linear and quadratic heat loss coefficients a_1 and a_2 . The flat plate and vacuum tube solar collectors available on the market were used with optical efficiency η_0 of 0.82 and 0.78, with a_1 at 3.61 and 1.48 W/m²K and a_2 at 0.014 and 0.008 W/m²K², respectively.

B. Absorption cooling unit

From the different types of absorption cooling machines, the water-cooled single effect model is that which is most commonly used for cooling applications for buildings.

The capacity of the absorption cooling unit depends on cooling load of the building. In turn, the cooling load of the building depends on ambient temperature, solar irradiation, internal gains, and thermal characteristics of building. Chosen absorbtion chiller working pair is LiBr-H₂O with a design generator entry temperature of 65°C at an absorber cooling water entry temperature of 26°C and an evaporator entry of 12°C and exit of 6°C.

| TABLE II | |
|---|-------------------------|
| SUMMARY OF SOLAR COOLING SYSTEM DESIGN VALUES FOR 1 | THE DIFFERENT BUILDINGS |

| Building case | Full cooling load hours (h) | Cooling energy demand (kWh) | Abs. chiller power (kW) | Collector area (m ²) | | Storage volume (m ³) | | |
|---------------|-----------------------------------|--------------------------------------|----------------------------|----------------------------------|----------------|----------------------------------|----------------|-----------------------------------|
| | | | | Flat plate | Vacuum tube | Flat plate | Vacuum tube | Average COP of absorption chiller |
| Build1_Low | 169 | 662.6 | 15 | 5 | 3 | 0.4 | 0.3 | 0.55 |
| Build1_High | 541 | 3355.9 | 25 | 26 | 17 | 2.1 | 1.3 | 0.55 |
| Build2_Low | 102 | 399.6 | 15 | 3 | 2 | 0.2 | 0.2 | 0.55 |
| Build2_High | 379 | 2400.9 | 25 | 18 | 12 | 1.5 | 1.0 | 0.55 |

V.ECONOMICAL ANALYSIS

To plan and design solar cooling systems, economic considerations and analyses make the basis for decision makers.

The costs in energy economics can be divided into three categories:

- capital costs, which contain the initial investment including installation,
- operating costs for maintenance and system operation,
- costs for energy and other material inputs into the system.

The total costs per kWh of cold produced C_{sum} are obtained by summing the chiller cost $C_{chiller}$ to the solar thermal system costs C_{solar} , the auxiliary heating costs $C_{heating}$, and costs for the cooling water production C_{cool} . The costs for heating have to be divided by the average COP of the system to refer the cost per kWh heat to the cold production and multiplied by the solar fraction for the respective contributions of solar and auxiliary heating. For the cooling water, costs per kWh of cooling water were taken from the literature review [12,27] and referred to the kWh of cold by multiplication with (1+(1/COP)) for removing the evaporator heat (factor 1) and the generator heat with a factor 1/COP.



Fig. 5. Cooling costs per kWh of cold for building with different internal gains.

A value for C_{cool} of 0.009 \in/kWh cooling water was used and auxiliary heating costs $C_{heating}$ were set to 0.04 \in/kWh heat.

To evaluate the benefits of a solar cooling system in comparison with an electrically driven cooling system, two different price scenarios were considered:

- electricity price increase at the same rate as the social discount rate, that is 3.5 %;
- electricity price increase at the current rate, which is 10%.



Fig. 6. Benefits from solar cooling system compared with electrically driven cooling system for different electricity price increase scenarios.

VI. ENVIRONMENTAL ANALYSES

The environmental analysis of the solar cooling system comprises an analysis of green house gasses emissions, ozone depletion substance emissions and water consumption and pollution. For its part, the environmental impact of energy use in solar cooling systems can be measured by kg of greenhouse gases emitted per kWh of consumed energy (t_{CO2} /kWh). The cooling equipment might release greenhouse gases into the atmosphere in two ways:

- Directly: refrigerant may be released into the atmosphere during equipment installation, normal operation, decommissioning, or eventual disposal.
- Indirectly: due to the fact that air-conditioning equipment use electricity (necessary to operate the system) generated from different fossil fuels, nuclear power and water, it releases greenhouse gases.

Direct CO_2 emissions were assumed insignificant due to the high quality and security of solar cooling systems manufacturing and installation. Solar thermal systems produce negligible environmental pollution during their manufacture, operation and dismantling. Solar cooling systems have the advantage of using harmless working fluids such as water or solutions of certain salts; they are totally environmentally safe.

The extent of the carbon dioxide emissions for a given carbon content in the fuel and for a given generation efficiency depends primarily on the energy efficiency of the solar cooling equipment.



Fig. 7. Reduction of CO₂ and potential return of investments due to CO₂ trading (15 \notin/t_{CO2}).

The amount of generated carbon dioxide is directly proportional to the amount of used energy or, in case of the solar cooling system, reduced electricity consumption to cover cooling load of the building. The analysis that can determine the overall contribution to global warming from energy using equipment over its operating lifetime is the Total Environmental Warming Impact (TEWI) analysis [28]. TEWI is an index that should be used to compare the global warming effects of alternative air-conditioning systems because it includes these contributions from the refrigerant, cooling efficiency, and weight.

A 20-year lifetime was assumed for the solar cooling system. The TEWI factor is expressed in terms of equivalent kg CO_2 direct and indirect emissions.

$$TEWI = T_{directCO2} + T_{indirectCO2}$$
(2)

In terms of cost per ton of CO_2 saved per year and for the lifetime of the solar cooling system, the results vary depending

on the building and on the rate of increase in fuel price (see Table III).

It was assumed that generation of electricity is from gas in condensation mode because it is typical for Latvia. The emission factor was taken 0.365 t_{CO2}/MWh_{el} .

| THE SUMMARY OF ENVIRONMENTAL ANALYSES OF SOLAR COOLING SYSTEM IN DIFFERENT BUILDINGS | | | | | | | | | | |
|--|--------------------------------------|--------|------------------------------|---------|--|---------|--|---------------------------------------|--|--|
| Building case | Reduction of electricity consumption | | Reduction of CO ₂ | | Reduction of CO ₂ per lifetime of 20 years (15 EUR/t _{CO2}) | | Money savings per lifetime of 20 years in terms of electricity consumption | | | |
| | (kWh/a) | EUR/a | (tonnes/a) | EUR/a | tones | EUR | Electricity price increase 3.5 % | Electricity price increase 10 % | | |
| | | | | | | | EUR per lifetime | EUR per lifetime | | |
| Build1_Low | 220.9 | 21.42 | 87.7 | 1315.17 | 1753.6 | 26,303 | 585.9 | 878.3 | | |
| Build1_High | 1118.6 | 108.51 | 444.1 | 6661.49 | 8882.0 | 133,230 | 2967.7 | 4448.8 | | |
| Build2_Low | 133.2 | 12.92 | 52.9 | 793.29 | 1057.7 | 15,866 | 353.4 | 529.8 | | |
| Build2_High | 800.3 | 77.63 | 317.7 | 4765.70 | 6354.3 | 95,314 | 2123.1 | 3182.7 | | |
| Build3_Ref | 1566.7 | 151.97 | 622.0 | 9329.50 | 12,439.3 | 186,590 | 4156.3 | 6230.6 | | |

TABLE III

VII. CONCLUSIONS AND DISCUSSION

Economic and environmental aspects of a solar cooling system in different buildings were analyzed in the paper. An absorption chiller was selected to maintain internal room temperature levels at a given point of 24°C for all operation hours. Different cooling load files with low and high internal gains for different building cases were developed using the calculation tool *CooL*. Analyses showed that in order to achieve a given solar fraction of the total heat demand for cooling mostly different solar collector areas and storage volumes are required, depending on the building load file and chosen collector surface area is in the range of 3 to 26 m², however the aperture area is around 1.3 m²/kW, if the generator is operated at a constant high temperature of 65°C.

For buildings with different internal gains and envelope characteristics, the required collector area varies by a factor of 4 to get the same solar fraction. The total system costs for commercially available solar cooling systems are between 1.7 and $2.5 \notin kWh$, depending on the cooling load of thebuilding and the chosen collector type. The total costs are dominated by the costs for the solar thermal system and the absorption cooling machine.

It was found that solar cooling systems in climate conditions of Latvia have the potential to reduce CO_2 emissions, however the economic feasibility and attractiveness is still low even though the price of electricity may increase even up to 10% for the solar cooling lifetime.

Solar cooling requires higher investments, but is possible to reduce costs if the solar system is designed for both needs – cooling in the summer and heating in the winter.

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Dzintars Jaunzems, Ivars Veidenbergs. Ar saules enerģiju darbināma aukstuma iekārta Latvijas klimatiskajos apstākļos: vides un ekonomiskie aspekti

Saistībā ar to, ka modernajā ēku arhitektūrā arvien vairāk tiek izmantotas stiklotās ārējās norobežojošas konstrukcijas, nepārtraukti tiek paaugstinātas prasības pret iekštelpu mikroklimatu, kā arī pieaugošais dažādu elektrisko un elektronisko ierīču izmantošanas apjoms var izraisīt ēku aukstuma slodzes palielināšanos. Ar saules enerģiju darbināma absorbcijas tipa dzesēšanas iekārta ir viens no risinājumiem kā efektīvi un videi draudzīgā veidā nosegt ēkas aukstuma slodzi, jo saules starojuma intensitāte praktiski sakrīt ar ēkas aukstuma slodzi. Darbā ir izvērtēti ar saules enerģiju darbināmu dzesēšanas iekārtu ekonomiskie un vides aspekti ēkām ar dažādiem iekšējiem siltuma ieguvumiem un ārējo norobežojošo konstrukciju parametriem. Lai sasniegtu saules enerģijas daļu vismaz 60%, nepieciešamo saules kolektoru virsma ir robežās no 3 līdz 26 m², savukārt īpatnējā saules kolektoru virsma ir robežās no 3 līdz 26 m², savukārt īpatnējā saules kolektoru virsma ir robežās no 1.7 līdz 2.5 \notin/kWh atkaītībā no ēkas iekšējiem situma ieguvumiem un ēkas ārējo norobežojošo konstrukciju energoefektivitātes rādītājiem. Ar saules enerģiju darbināma absorbcijas tipa dzesēšanas iekārta ģeneratoram tiek nodrošināts siltums ar konstantu temperatūru 65°C. Aukstuma enerģijas izmaksas ir robežās no 1.7 līdz 2.5 \notin/kWh atkaītībā no ēkas iekārtai ir pietiekoši liels CO₂ samazinājuma potenciāls, tomēr neskatoties uz to, šādu dzesēšanas sistēmu izmaksas joprojām ir ekonomiski nepamatotas un nav konkurētspējīgas, ja salīdzina ar kompresijas tipa dzesēšanas iekārtām, kas tiek darbinātas ar elektroenerģiju, t.sk. ņemot vērā elektroenerģijas tarifa paaugstināšanos jau tuvākajā nākotnē.

Дзинтарс Яунземс, Иварс Вейденбергс. Системы охлаждения малой мощности на основе солнечной энергии для климатических условий Латвии: экологические и экономические аспекты

В связи с тем, что современная архитектура зданий все чаще использует застекленный дизайн, постоянное увеличение требований к внутреннему микроклимату, а также повышение объема использования различных электротехнических и электронных устройств в зданиях может вызвать увеличения нагрузки охлаждения. Система охлаждения на основе солнечной энергии является эффективным и экологически чистым способом компенсации охлаждающей нагрузки здания, так как интенсивность солнечного излучения практически совпадает с нагрузкой на систему охлаждения. В работе рассмотрены экономические и экологические аспекты системы охлаждения на основе солнечной энергия, необходима поверхность солнечного излучения практически совпадает с нагрузкой на систему охлаждения. В работе рассмотрены экономические и экологические аспекты системы охлаждения на основе солнечной энергии в зданиях с различными внутренними и внешними параметрами. Для достижения по крайней мере 60% солнечной энергия, необходима поверхность солнечного коллектора от 3 до 26 м² с удельной площадью поверхности солнечного коллектора около 1.3 m²/кВт, если поглощение тепла генератором осуществляется при постоянной температуре 65°С. Расходы на энергию в диапазоне от 1,7 до 2,5 €kВтч в зависимости от внутренних источников энергии в здании. Системы охлаждения малой мощности на основе солнечной энергии имеют достаточно большие возможности для сокращения выбросов CO₂, но несмотря на это, они все еще остаются экономически неоправданными и не являются конкурентоспособными по сравнению с традиционными