

THE INFLUENCE OF TEMPERATURE CHANGES ON THE EIGEN -FREQUENCIES AND MODE SHAPES FOR RECTANGULAR PLATES CLAMPED ON THE CONTOUR

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Abstract: The alysis of free vibrations on rectangular plates with mixed conditions of buckling in the thermal environment is carried out by means of the 3D elasticity theory. In this paper there has been analyzed by the finite element method (FEM), the mode shape and the eigen-frequencies for a rectangular plate clamped on the contour for different temperatures. In this analysis successive heat degrees have been used, this fact helping to find the critical values in which the frequencies were at zero and to correlate these temperatures with the values obtained from the static analysis The temperature rise, the material graded index and the geometrical parameters on the eigen-frequencies were studied.

Keywords: temperature changes, eigen-frequencies, mode shapes, rectangular plates, FEM

1. Introduction

The plates are generally defined as being solids; the thickness is less than the length and respectively the width and is used in various fields as well as in engineering.[1-3]. Thus, numerous studies in engineering focused on the study of static and dynamic behavior of the plates, being established how the deformation of these plates takes place. The dynamic behavior of the plates is defined by a few basic elements, which change their shape in particular and this leads to changes in the vibration modes and eigen frequencies [4]. The mechanic and physic characteristics of the material the plates are built off are the following [5-10]: the longitudinal elasticity module or the Young module (noted with E); the transversal elasticity module (G); the Poisson coefficient (v); the density (ρ). The minimal values of these parameters for steel plates are given in table 1:

Tab. 1 The mechanic an	nd physic characteristics
	of plates

Chara	acteristic	Values
The	density	78007900[kg/m ³]
The	Longitudinal	$2.1 \times 10^{10} [\text{N/m}^2]$
elasticity module	Transversal	$(7.367.93)*10^{10}[N/m^2]$
The Poiss	on coefficent	0.27-0.30 [-]

The relation existing between these parameters [11]: $G = \frac{E}{2(1+\nu)}$

In this paper there are modally plate-shaped plates, whose is propping is done by clamped edges on all sides. The results from the simulation tables and charts, highlighting the influence of temperature changes of their eigen frequency and vibration modes are shown.

1.1 The modal analysis of plates having different temperatures

The chosen method for the modal analysis

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is the finite element method (FEM) using the SolidWorksprogram in the Geometry module.The 3D model used for the 42 cases, is a thin rectangular elastic plates with the sizes of 1000 x 500x 2 [mm]for length, width and respectively thickness.

In the modal analysis of the plates at different temperatures, for obtaining the eigen frequencies and the vibration modes, one has used the static analysis module und the action of its own weight[12-14]: the stages for the modal analysis are; creating the plates geometry; creating the thermal analysis study respectively the frequency type; choosing the material; applying the restraints (figure 1); discretion of finite elements (figure 2); running the modal analysis calculation; visualizing the results.

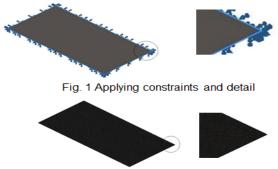


Fig. 2 Discretion of finite elements and detail Following the simulation, there were determined the first 30 mode shapes for the 42 cases examined. The mesh 3D model was made with tetrahedral finite elements of the finite element with an average size of 2 mm, and the material used in the database of the program was the 1.023 Carbon Steel, the mechanical properties being shown in table 2:

Tab. 2 The material properties for the 1.023 Carbon Steel

Elastic Modulus	Shear Modulus	Mass Density	Poisson's Coefficient
[N/mm]	2 [N/mm]	3 [kg/m]	[-]
205000	80000	7858	0.29

Mesh in finite elements has been done by the "Solid mesh" option, using the following options: High for Mesh Quality, Solid mesh for Mesh type, Curvature based mesh for Mesher Used [15-17].The 42 cases, represent 42 different values of temperature and are: -50, -45, -40, -35, -30, -25, -20, -15, -10, -5, 0, 5, 10, 15, 20, 21, 22, 23, 24, 24.85, 25, 26, 27, 28, 29, 30, 31, 32, 33, 34, 35, 40, 45, 50, 55, 60, 65, 70, 75, 100, 500 şi 1400 [°C].

Of these positive and negative values of temperature, in table 6, there is summarized the mesh only the plate having a temperature of 24.85 (reference temperature), and in figure 3 there is show the vibration mode shapes.

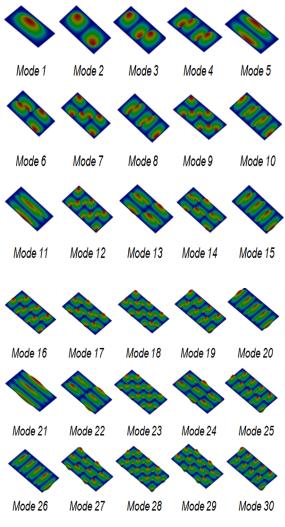


Fig. 3 Mode shapes for the rectangular plate

2. Plates simuation results

The results obtained by FEM for the eigen frequencies according to the rectangular plate vibration modes at the temperature of $1400 \div -50^{\circ}$ C, are numerically presented in table $3\div 6$. Also, for the other 42 cases, one has obtained certain eigen frequencies values.

These results are graphically presented in

figure 4 to 15, as follow:

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Mode	1400	500	100	75	60	70	65	55	45	50
1	1840.6	1301.5	461.24	387.48	301.7	363.25	323.12	285.18	209.26	248.08
2	1833.9	1273.1	458.88	384.7	298.78	357.47	312.77	281.73	208.1	247.42
3	1818.1	1260.5	450.39	379.4	298.18	346.58	309.59	274.2	201.07	244.15
4	1767.3	1177.9	437.32	374.67	294.69	339.98	309.19	258.32	200.8	243.94
5	1734.5	1123.2	433.81	348.35	284.39	335.1	307.32	250.32	189.9	243.04
6	1636.7	1081.2	378.31	326.38	267.79	333.5	305.48	240.32	188.97	241.83
7	1591.2	1022.7	339.48	312.32	258	332.73	301.06	233.56	188.88	240.09
8	1496.8	1007.3	330.62	294.6	253.3	325.23	292.54	229.14	186.79	232.09
9	1387.5	948	326.68	293.17	241.88	315.54	281.34	218.86	186.7	230.85

Table 3(eigen frequencies according to the rectangular plate vibration modes at the 1400÷ 55°C)

Table4(eigen frequencies according to the rectangular plate vibration modes at the $25 \div 35^{\circ}C$ *)*

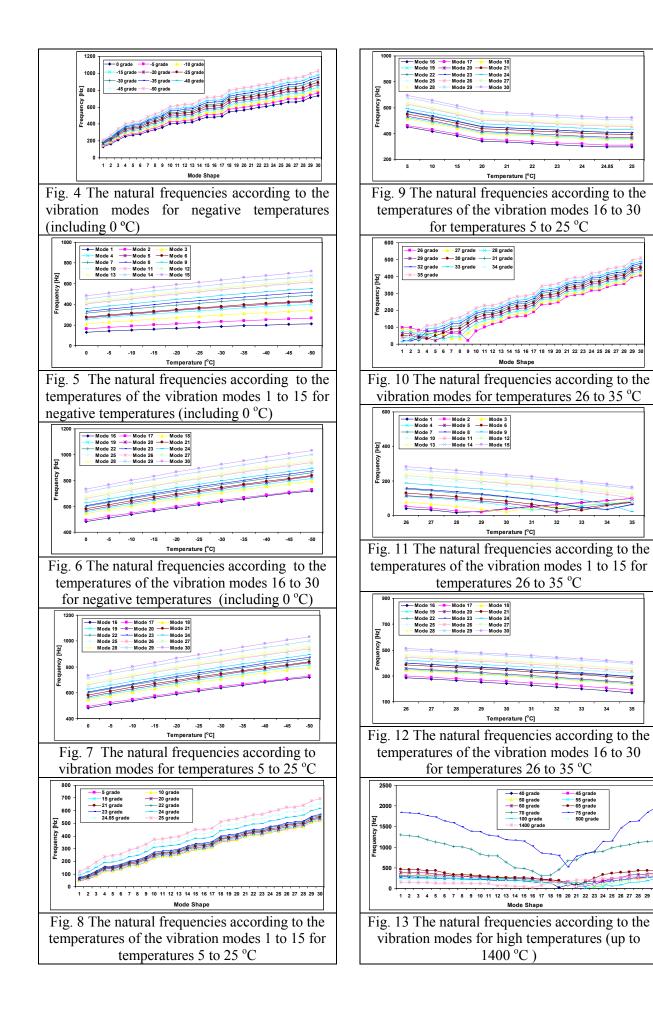
Mode	25	24.85	24	23	22	21	20	15	10	5
1	46.892	47.911	53.281	58.919	64.013	68.692	73.041	73.041	106.46	119.43
2	60.829	62.036	68.444	75.235	81.416	87.124	92.451	92.451	133.84	150.03
3	85.927	87.266	94.473	102.27	109.48	116.22	122.56	122.56	173.05	193.15
4	122.05	123.46	131.17	139.23	146.41	153.24	159.75	159.75	213.56	235.59
5	123.5	124.76	131.62	139.67	147.66	155.23	162.42	162.42	221.4	245.41
6	137.28	138.59	145.76	153.75	161.31	168.51	175.4	175.4	232.7	256.3
7	161.01	162.38	169.92	178.35	186.39	194.07	201.44	201.44	263.5	289.31
8	168.67	170.12	178.14	187.13	195.69	203.88	211.74	211.74	278.03	305.65
9	195.17	196.59	204.44	213.29	221.78	229.93	237.79	237.79	304.99	333.32

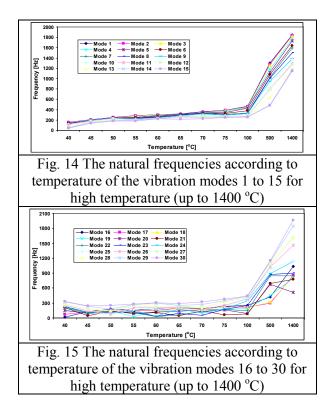
Table 5 (eigen frequencies according to the rectangular plate vibration modes at the $0 \div 24.85^{\circ}C$ *)*

	10	<u> </u>		U						
Mode	40	35	34	33	32	31	30	29	28	27
1	154.09	97.633	86.485	75.323	62.985	50.88	38.131	22.251	14.515	29.808
2	152.72	97.093	85.152	72.959	61.899	48.984	35.711	19.336	25.945	41.202
3	143.02	81.038	71.333	58.162	50.328	40.679	20.168	33.644	52.005	65.371
4	141.91	78.98	65.758	58.07	36.093	30.868	57.165	74.824	89.053	101.28
5	135.18	77.083	63.138	42.069	20.482	50.856	68.803	82.852	94.764	105.27
6	134.19	74.802	58.007	28.629	41.224	64.779	81.695	95.578	107.62	118.38
7	132.83	64.993	33.703	43.975	70.602	89.542	105.05	118.48	130.49	141.43
8	131.98	62.398	32.451	45.992	73.284	93.047	109.35	123.53	136.24	147.86
9	118.85	21.765	65.505	89.971	109.02	125.15	139.38	152.25	164.08	175.09

Table 6 (eigen frequencies according to the rectangular plate vibration modes at the $-50 \div -5^{\circ}C$)

Mode	-50	-45	-40	-30	-25	-20	-15	-10	-5	0
1	213.15	206.5	199.61	185.02	177.24	169.08	160.48	151.35	141.58	131.01
2	267.81	259.42	250.75	232.38	222.59	212.33	201.51	190.03	177.77	164.53
3	341.37	330.76	319.79	296.57	284.21	271.27	257.64	243.21	227.82	211.24
4	401.88	389.87	377.45	351.22	337.29	322.72	307.41	291.24	274.04	255.6
5	425.88	412.86	399.4	370.96	355.86	340.05	323.44	305.89	287.21	267.17
6	435.47	422.49	409.08	380.77	365.74	350.03	333.52	316.1	297.6	277.77
7	487.34	472.93	458.05	426.65	410	392.61	374.36	355.12	334.71	312.89
8	518	502.53	486.55	452.85	434.99	416.33	396.77	376.14	354.27	330.89
9	553.75	537.61	520.95	485.83	467.24	447.83	427.51	406.11	383.46	359.31





From the tables and graphs above, it is noted that the natural frequencies of the rectangular plate embedded in the contour fall to negative temperature to 28°C for mode one, 29°C for mode 2, 30°C for mode 3 and so on, and then increase with the temperature.

From Tables 3 to 6 and Figures 4 to 15 it is also observed that the frequencies increase approximately linearly depending on the mode shape for negative temperatures and for positive temperatures up to the temperature of 28 the custom frequencies of the rectangular plate embedded on the contour increase approximately linearly depending on your own vibration modes.

From the 29°C frequencies to the first mode shape and then increase, for the over temperatures above 100°C have a very pronounced increase depending on the temperature.

3.Conclusion

The following conclusions can be drawn :

- For negative values of temperature, the eigen frequencies, and temperatures decrease linearly with an increase in the vibration modes;

- For positive temperatures, lower or near the reference temperature of 24.85°C, the eigen frequencies decline with the increasing temperature and with their own ways;

- For temperatures higher than the reference temperature (24.85°C), but up to 35°C, the eigen frequencies for the first vibration modes, present an anomaly, i.e. decrease and then increase to certain values, which does not happen but for the ways of large vibration;

- For high temperature applied evenly on the plate, the eigen frequencies increase sharply due to the change in the material it is constructed like.

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